EFFECT OF SELECTED COMPONENTS ON COMBUSTION OF METHANE

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Abstract: Article discusses the effect of various gases on the combustion of methane in LGW 702 combustion engine intended for use in cogeneration unit. The measured results were evaluated for internal, performance and economic parameters of the combustion engine the ideal operating parameters can be set. The aim of the study was to assess the effect of combustible carbon monoxide and carbon dioxide and nitrogen on the parameters.

KEYWORDS: methane, carbon dioxide, carbon monoxide, nitrogen, spark ignition engine

1. Introduction

As the whole world tries to lower the emissions of greenhouse gases the effective use of other than conventional energy sources are gaining foothold. Municipal solid waste, its gasification and following energy production is considered such source of energy. Non-separable waste can be turned to syngas through various gasification processes. The most common processes are direct and indirect gasification and pyrolysis. [1,2] By employing such treatment gas, tars/oils and char are created. Controlling the amount of tars and particulate matter are of highest priority in combustion engines. This means, that the produced syngas needs to be cleaned prior to combustion. If left untreated, injector nozzles may become blocked by particulate matter. Therefore, the amount of PM shouldn't exceed 50mg/Nm³ and size of 10µm. Below the 100°C most of the tars are condensed, therefore the hot gas should be filtered for tars. If the excess of tars are being present in the gas powering the engine, the pipes of the engine, as well as valves may become clogged [3,4,5,6].

The cleaned gas, which is the main interest of this paper, consists mainly of hydrogen (H₂), carbon monoxide (CO), carbon dioxide (CO₂), nitrogen (N₂) and methane (CH₄). The inert components tend to reduce maximum in-cylinder pressure as more of the space inside the cylinder is occupied by the non-combustible gas, therefore, overall stress on crank mechanism is reduced. [7] The content of each component in the gas is dependant mainly on the type of feedstock and type of gasifying process used. As the non-separable municipal waste is continuously fed to the reactor, the produced gas composition is changing with time due to non-uniformity of the waste. By finding the optimal ignition angle for various mixtures, it is possible to understand the influence of various gas components on the combustion. Such syngases can be also be used in dual-fuel compression ignition engines where the lean mixture is ignited by various types of bio-oils or oil from pyrolysis [8,9].

This article represents the partial study of the influence of inert gases (CO₂ and N₂) and carbon monoxide on combustion of methane and natural gas. The natural gas at the time of measurement consisted of 98% of methane, therefore it was assumed, that the results should be consistent with those of pure methane. The in-cylinder pressure used to evaluate the start of

combustion (SOC) and end of combustion (EOC) was measured with Kistler 6117 BDC15 spark plug coupled with Kistler 2614A rotational encoder used to determine the exact position of crankshaft. Spark plugs were connected to charge-amplifier coupled with A/D converter. The engine used in the experiment was modified Lombardini LGW 702, twin cylinder, water-cooled, four-stroke engine. The original pistons were machined to reach compression ratio of 12.5:1 from the former ratio of 22.8. The fresh charger was prepared in mixer with a difusser. The whole measuring stand can be found on the following figure (Fig. 1).

Fig. 1 Basic scheme of measuring for LGW 702 engine (1 - intake manifold, 2 – Kistler rotational encoder, 3 – Exhaust tip, 4 – Exhaust temperature sensor, 5 – Kistler 6117 measuring spark plug, 6 – AVL dynamometer, 7 – *Stepper motor* richness regulation, 8 – Pressure bottles (CH_4 and $\text{CO/CO}_2/\text{N}_2$), 9 – Gas mass flow meter

Fig. 2 Engine ECU used in experiment

The gas was supplied from pressure bottles and entered the intake system through a gas diffuser. The regulation of the stoichiometry was carried out by a stepper motor controlled by a feedback controller. As the narrow-band lambda controllers only sense if the mixture is rich or lean, secondary wide band lambda controller was also employed to monitor the operation of the feedback system. Optimization of ignition timing was achieved by an IMF Soft Master ignition control unit. The timing signal was measured on camshaft fitted with 5 sensing elements. As the control unit possess direct TCI ignition output, a coil on plug system was chosen to deliver a spark energy of up to 65mJ through a BOSCH P-65T coil-on-plug ignition coils. The maximum of 65mJ energy was chosen based on a fact, that high content of inert gases tends to increase the minimum ignition energy. This ensures a successful ignition of all mixtures.

As carbon monoxide is combustible gas, 7 different measurements were done. First, clean CH₄ was evaluated. In consecutive runs, the content of CO was increased up to 100% CO. The effect of inert gases was measured on four different mixtures. The methane was diluted with 25% and 50% of inert gas for both, CO₂ and N₂. The chemical and physical properties of the mixtures can be found in the following table (Table 1).

Table 1 Physical and chemical parameters of mixtures used in experiment

Mixture		CH ₄ /CO							NG/CO ₂		NG/N ₂	
NG/CH ₄ content	[%vol]	100	63.4	29.4	13.9	4.9	2.2	0	75	50	75	50
CO/CO ₂ /N ₂ content	[%vol]	0	36.6	70.6	86.1	95.1	97.8	100	25	50	25	50
Density	[kg.m ⁻³]	0.72	0.881	1.049	1.09	1.134	1.15	1.16	1.216	1.509	0.8511	0.984
Molar mass	[kg.kmol ⁻¹]	16	20.39	24.47	26.33	27.41	27.73	28	23	30	19	22
Lower heating value	[MJ.kg ⁻¹]	50.03	29.96	17.78	13.47	11.24	10.61	10.11	26.11	13.34	31.601	17.94
AFR ratio	[-]	17.24	9.81	5.30	3.71	2.88	2.65	2.46	8.99	4.598	10.889	6.269
Mixture mass heating value	[MJ.kg ⁻¹]	2.74	2.77	2.82	2.86	2.90	2.91	2.92	2.612	2.384	2.658	2.503
Mixture volume heating value	[MJ.m ⁻³]	3.19	3.14	3.21	3.31	3.40	3.44	3.48	2.971	2.748	3.00	2.786

2. Experimental

Optimal ignition angle was controlled manually through a PC software supplied with the ECU. The measurements conditions were 1500 min⁻¹, full engine load and stoichiometric mixture. All measurements were performed at full load. The results were measured and compared to reference fuel (CH₄). In case of all fuels and mixtures, pressure traces of only the optimal ignition angle of methane (24.4°CA BTDC) were evaluated.

The duration of combustion (10-90MFB) for methane lasted 24.4° CA. Pure carbon monoxide had a combustion duration of 30.7°CA. The dependence of combustion period on content of the species is not linear. By mixing 5% to around 15% vol of CH₄ with CO the main combustion period was shortened to 17° CA. This is caused by the introduction of hydrogen containing species which

enhance the oxidation of CO. The HO_2 , H and OH all speed increases the oxidation rates of CO. The ignition delay for such mixture is also the shortest, lasting only 15° CA. Value for methane is 22° CA and 18° CA for CO.

The maximum engine in-cylinder pressure reached lowest value of 4.7 MPa while running on carbon monoxide. By increasing the content of CH_4 , the pressure steadily increased and reached a maximum value of 7.1 MPa at 13.9% of CH_4 . By further increasing the CH_4 content the maximum in cylinder pressure decreased. The maximum pressure for pure methane was 5.6 MPa. The point of maximum pressure was between 374.4° and 369.1° CA for constant ignition angle of 24° CA as can be seen from Fig. 4.

Fig. 3 Course of in-cylinder heat release for various CO and CH_4 mixtures

Fig. 4 Traces of pressure for various CO and CH₄ mixtures between 270 and 450° CA

Fig. 5 Course of in-cylinder heat release for natural gas and inert gases

With a change of mixture of combustibles to a mixture of natural gas and inert gas, the burning duration generally prolonged compared to those of methane and carbon monoxide. As mentioned above, the reference of methane was considered equal with natural gas, as it

contained 98% methane at the time of measurement. This was also confirmed by the duration of combustion phase which was only 0.1° CA shorter than for CH_4 . Peak pressures varied by only 0.012 MPa and were shifted by 0.7° CA as can be seen from Figure 6. The main combustion phase was the shortest for mixture of 25% N_2 with a value of 24.9° CA. With the same amount of carbon dioxide, the phase prolonged by 3.4° CA to 28.3° CA. Mixtures that contained 50% of either of the inert gases, the phase was 31.5° CA for N_2 and 36.3° CA for CO_2 . The point of 50% MFB of mixtures moved between 23.2° CA and 10.7° CA ATDC. Pure natural gas had a value of 8.7° CA ATDC as can be seen from Figure 5. It can be assumed from the results that the excess of nitrogen is a more favourable candidate than carbon dioxide due to a higher burning velocity.

As the inert gases are not reactive, less heat is added to the cycle and the peak pressure are also marginally lower than those of pure natural gas. The effect of addition of inert gas also caused the shift of the point of maximum pressure but in the opposite direction. This was accounted to the smaller and slower net heat release and pressure rise which was dominated by the pressure drop from gas expansion. The peak pressure for pure natural gas was 6.029 MPa at the 373.5° CA. By mixing 25% of carbon monoxide to the natural gas, the peak pressure moved by 2.1° CA to 375.6° CA with a value of 4.651 MPa. In case of nitrogen the shift was only -0.3° CA and reached a value of 5.461 MPa. The values of peak pressure for these mixtures were 4.65 MPa for CO_2 and 5.461 MPa for N_2 respectively. The mixtures of 50% inert gas caused the shift of the peak pressure point towards TDC as mentioned above. In case of 50% CO_2 , the peak pressure reaches value of 2.98 MPa at the point of 369.6° CA, half of pure natural gas. The declined of peak pressure for 50% N_2 was not so high as with the CO_2 . For such mixture, it occurred at 374.2° CA. The pressure at this point was 4.467 MPa as can be seen from the following Fig. 6.

Fig. 6 Traces of pressure for mixtures of natural gas and inert gases between 270 and 450° CA

CONCLUSION

By analyzing various mixtures at 1500 min^{-1} of methane with carbon monoxide and natural gas with inert gases, the following can be concluded:

- Indicated mean effective pressure increases by around 5.5% for CH_4 compared to CO
- By mixing 5 to 15% of CO to CH_4 the overall burning velocity and pressure rise rate increases ($0.326 \text{ MPa}/^\circ\text{CA}$). This can also be seen from maximum pressure in the cylinder (7.1 MPa)
- The volumetric heating value of the stoichiometric mixture gets lower with the growing share of CH_4 (3476 kJ.m^{-3} for CO and 3185 kJ.m^{-3})
- Greater pressure rise rate can be achieved by addition of CH_4 to the mixture ($0.139 \text{ MPa}/^\circ\text{CA}$ for CO and $0.191 \text{ MPa}/^\circ\text{CA}$ for CH_4)
- By introducing any carbon dioxide or nitrogen to the mixture, the maximum pressure decreases (by 50% for 50% of CO_2 and 30% for N_2)
- Addition of carbon dioxide or nitrogen extends all combustion phases
- The combustion duration for N_2 are smaller and peak pressures are higher and closer to TDC compared to addition of same amount of CO_2 . Therefore, the combustion with N_2 is more favourable than with CO_2 .

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